

# **Time Response Analysis**

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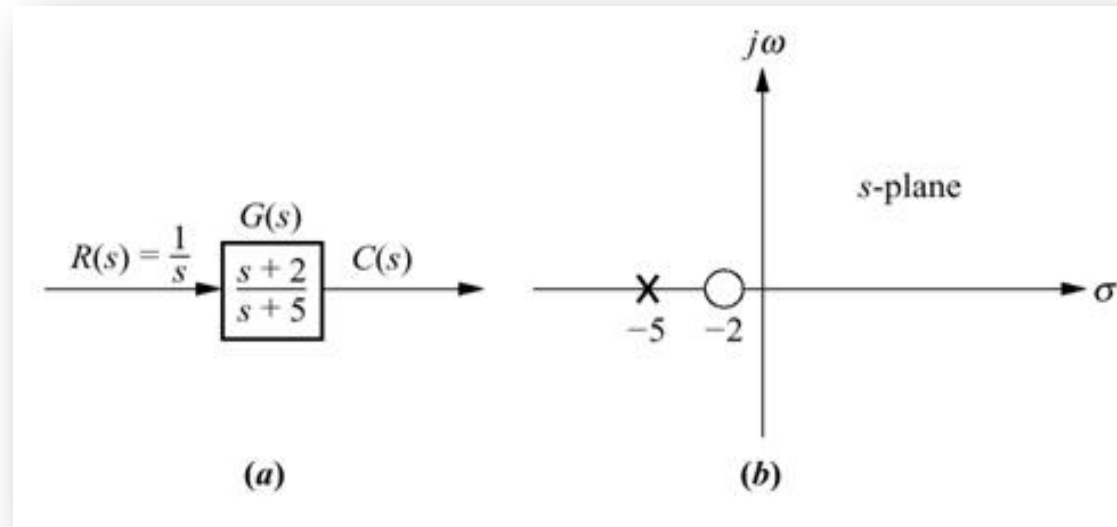
- ▶ Introduction
- ▶ Influence of Poles on Time Response
- ▶ Transient Response of First-Order System
- ▶ Transient Response of Second-Order System

# Introduction

- ▶ The concept of **poles** and **zeros**, fundamental to the analysis of and design of control system, **simplifies the evaluation of system response**.
- ▶ The **poles** of a transfer function are:
  - i. Values of the Laplace Transform variables  $s$ , that cause the transfer function to become **infinite**.
  - ii. Any **roots of the denominator** of the transfer function that are common to roots of the numerator.
- ▶ The **zeros** of a transfer function are:
  - i. The values of the Laplace Transform variable  $s$ , that cause the transfer function to become **zero**.
  - ii. Any **roots of the numerator** of the transfer function that are common to roots of the denominator.

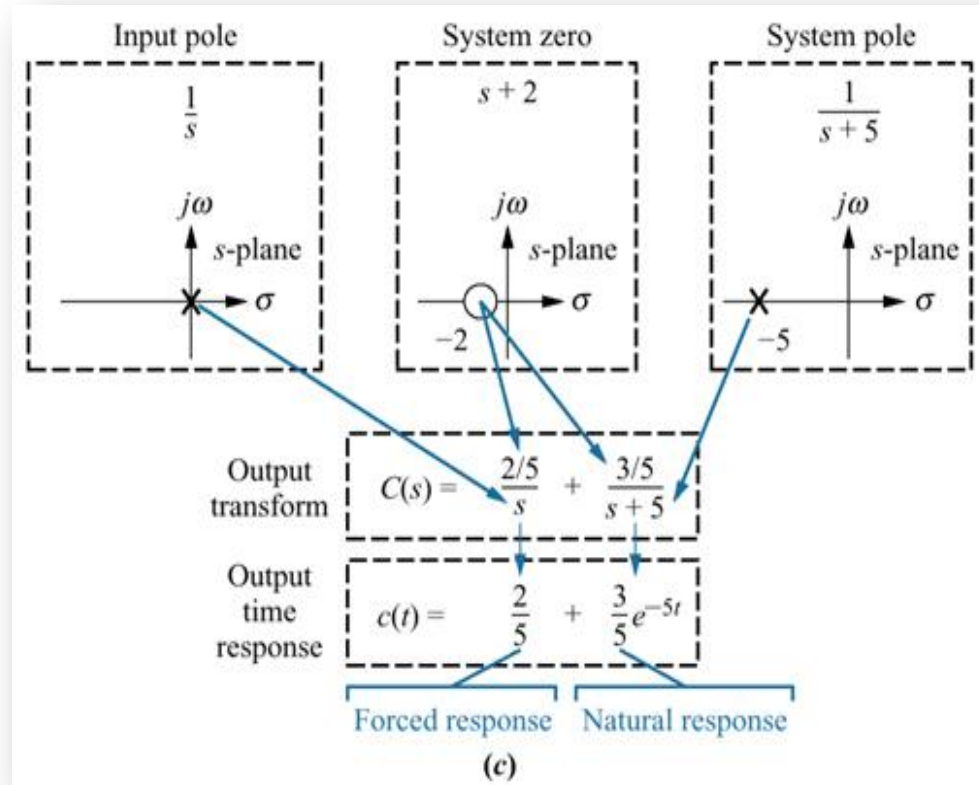
# Influence of Poles on Time Response

- ▶ The **output response** of a system is a sum of
  - Forced** response
  - Natural** response



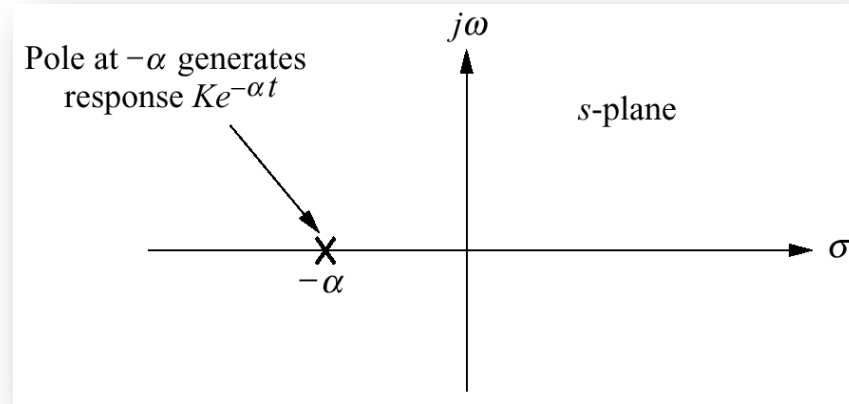
- System showing an input and an output
- Pole-zero plot of the system

# Influence of Poles on Time Response

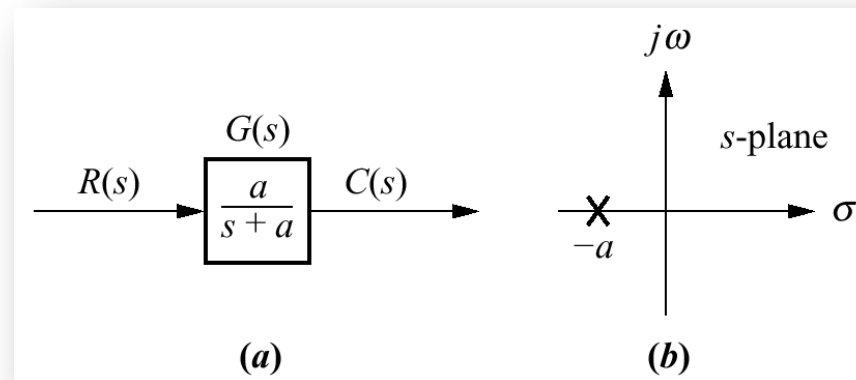


- c) Evolution of a system response. Follow the **blue arrows** to see the evolution of system component generated by the pole or zero

# Influence of Poles on Time Response



Effect of a real-axis pole upon transient response



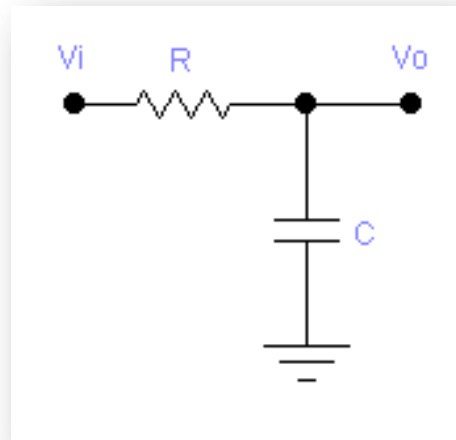
- a) First-order system
- b) Pole plot of the system

# First-Order System

- ▶ General form:

$$G(s) = \frac{C(s)}{R(s)} = \frac{K}{\tau s + 1}$$

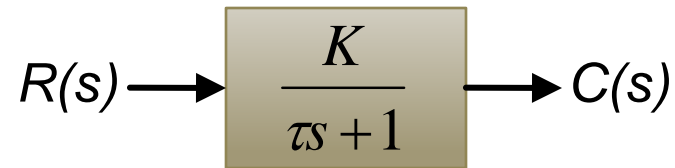
- ▶ Problem: Derive the transfer function for the following circuit



$$G(s) = \frac{1}{RCs + 1}$$

# First-Order System

- ▶ Transient Response: Gradual change of output from initial to the desired condition.
- ▶ Block diagram representation:



Where,

$K$  : Gain

$\tau$  : Time constant

- ▶ By definition itself, the input to the system should be a step function which is given by the following:

$$R(s) = \frac{1}{s}$$



# First-Order System

- ▶ General form:

$$\boxed{G(s) = \frac{C(s)}{R(s)} = \frac{K}{\tau s + 1}} \quad \longrightarrow \quad \boxed{C(s) = G(s)R(s)}$$

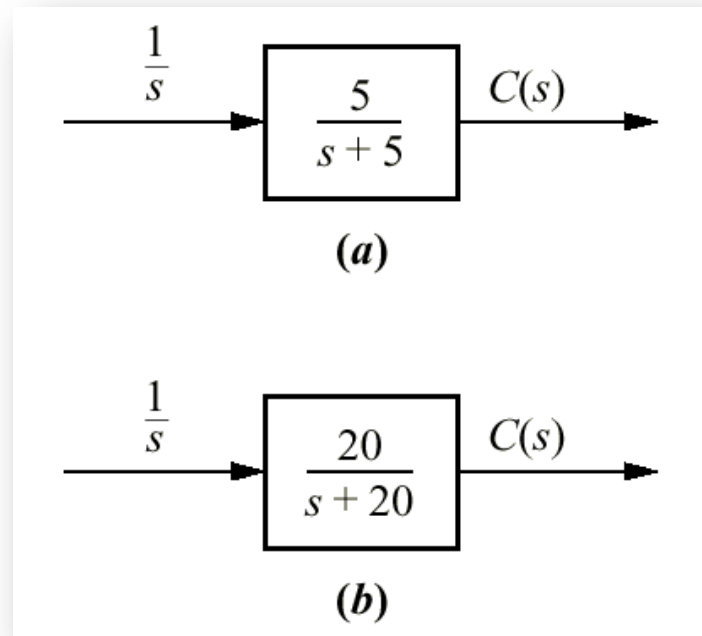
- ▶ Output response:

$$\begin{aligned} C(s) &= \left(\frac{1}{s}\right) \left(\frac{K}{\tau s + 1}\right) \\ &= \frac{A}{s} + \frac{B}{\tau s + 1} \end{aligned}$$

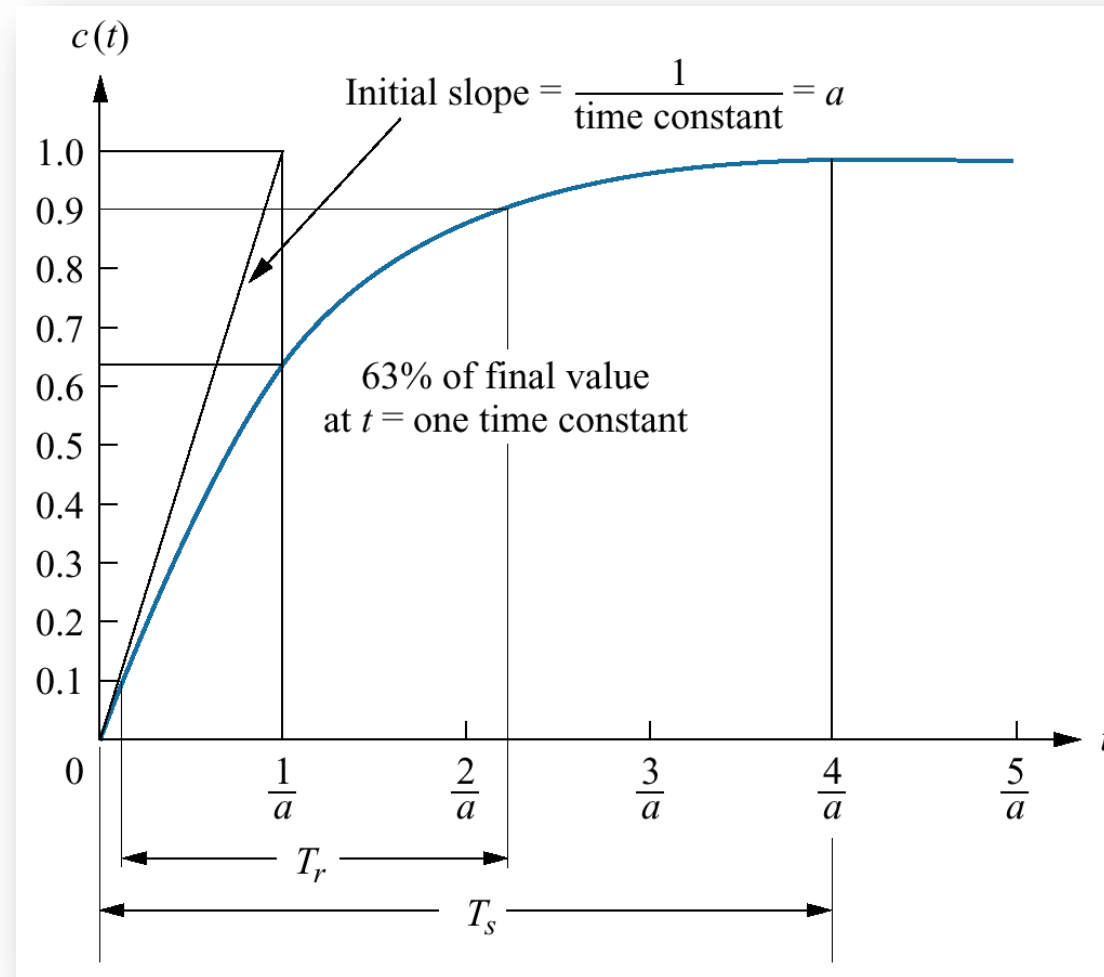
$$\boxed{c(t) = A + \frac{B}{\tau} e^{-t/\tau}}$$

# First-Order System

- Problem: Find the forced and natural responses for the following systems



# First Order System



First-order system response to a unit step

# Transient Response Specifications

- ▶ Time constant,  $\tau$

- ▶ The time for  $e^{-at}$  to decay 37% of its initial value.

$$\tau = \frac{1}{a}$$

- ▶ Rise time,  $t_r$

- ▶ The time for the waveform to go from 0.1 to 0.9 of its final value.

$$t_r = \frac{2.2}{a}$$

- ▶ Settling time,  $t_s$

- ▶ The time for the response to reach, and stay within 2% of its final value.

$$t_s = \frac{4}{a}$$

# Transient Response Specifications

- ▶ Problem: For a system with the transfer function shown below, find the relevant response specifications

$$G(s) = \frac{50}{s + 50}$$

- Time constant,  $\tau$
- Settling time,  $t_s$
- Rise time,  $t_r$

# Second-Order System

- ▶ General form:

$$G(s) = \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

Where,

K : Gain

$\zeta$  : Damping ratio

$\omega_n$  : Undamped natural frequency

- ▶ Roots of denominator:  $s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$

$$s_{1,2} = -\zeta\omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$

# Second-Order System

- ▶ Natural frequency,  $\omega_n$ 
  - ▶ Frequency of oscillation of the system without damping.
- ▶ Damping ratio,  $\zeta$ 
  - ▶ Quantity that compares the exponential decay frequency of the envelope to the natural frequency.

$$\zeta = \frac{\text{Exponential decay frequency}}{\text{Natural frequency (rad/s)}}$$

# Second-Order System

- ▶ Problem: Find the step response for the following transfer function

$$G(s) = \frac{225}{s^2 + 30s + 225}$$

- ▶ Answer:

$$c(t) = 1 - e^{-15t} - 15te^{-15t}$$



# Second-Order System

- ✖ Problem: For each of the transfer function, find the values of  $\zeta$  and  $\omega_n$ , as well as characterize the nature of the response.

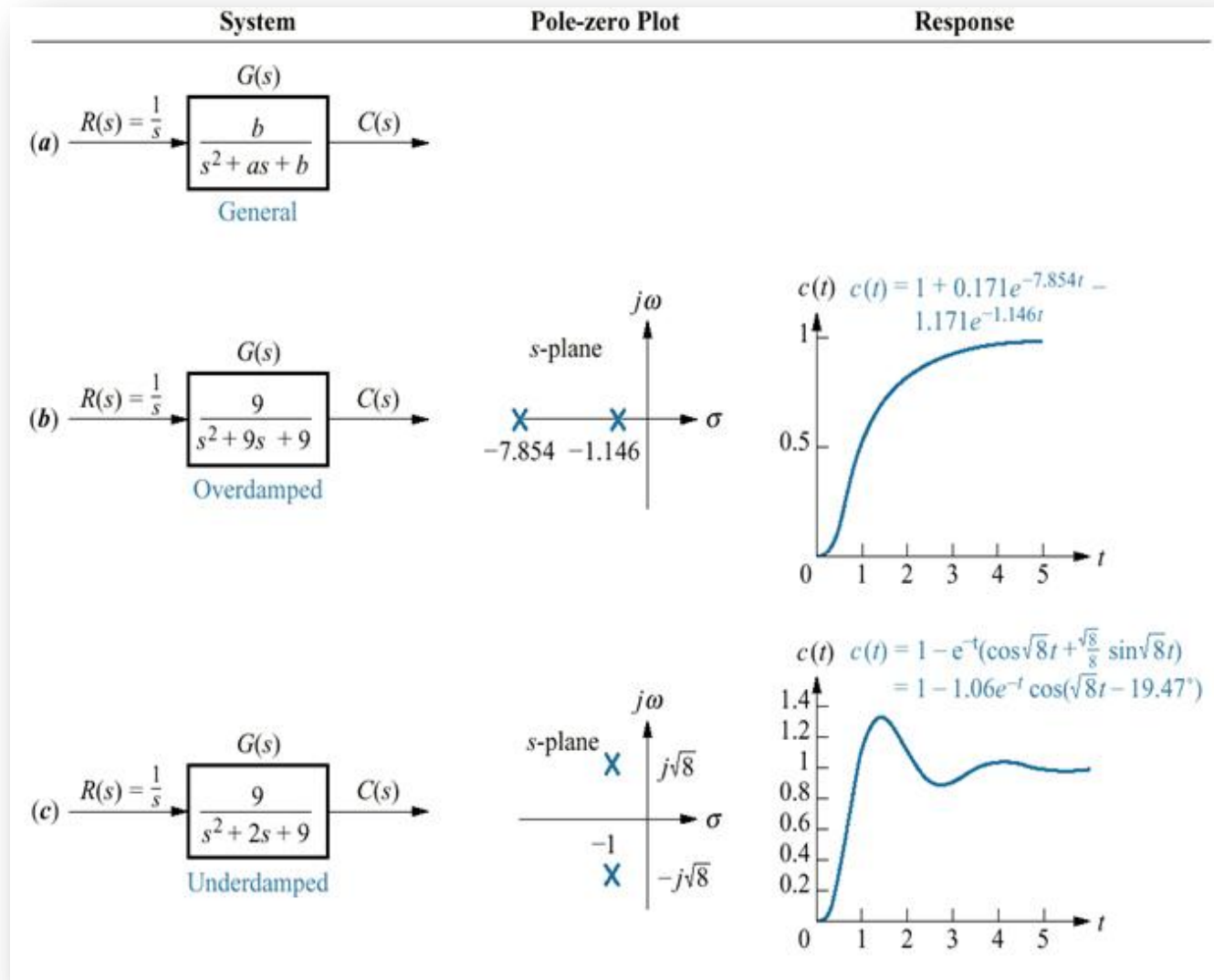
a) 
$$G(s) = \frac{400}{s^2 + 12s + 400}$$

b) 
$$G(s) = \frac{900}{s^2 + 90s + 900}$$

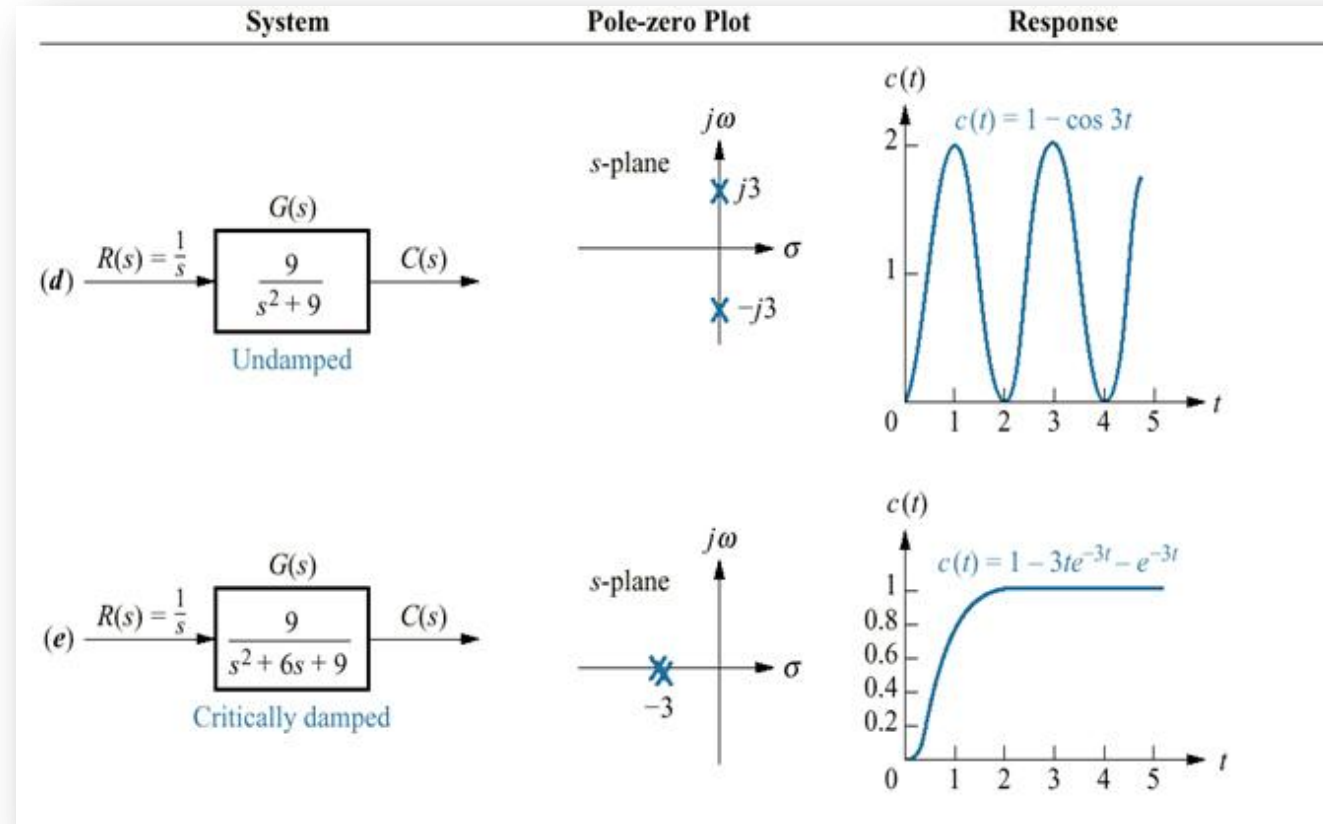
c) 
$$G(s) = \frac{225}{s^2 + 30s + 225}$$

d) 
$$G(s) = \frac{625}{s^2 + 625}$$

# Second-Order System

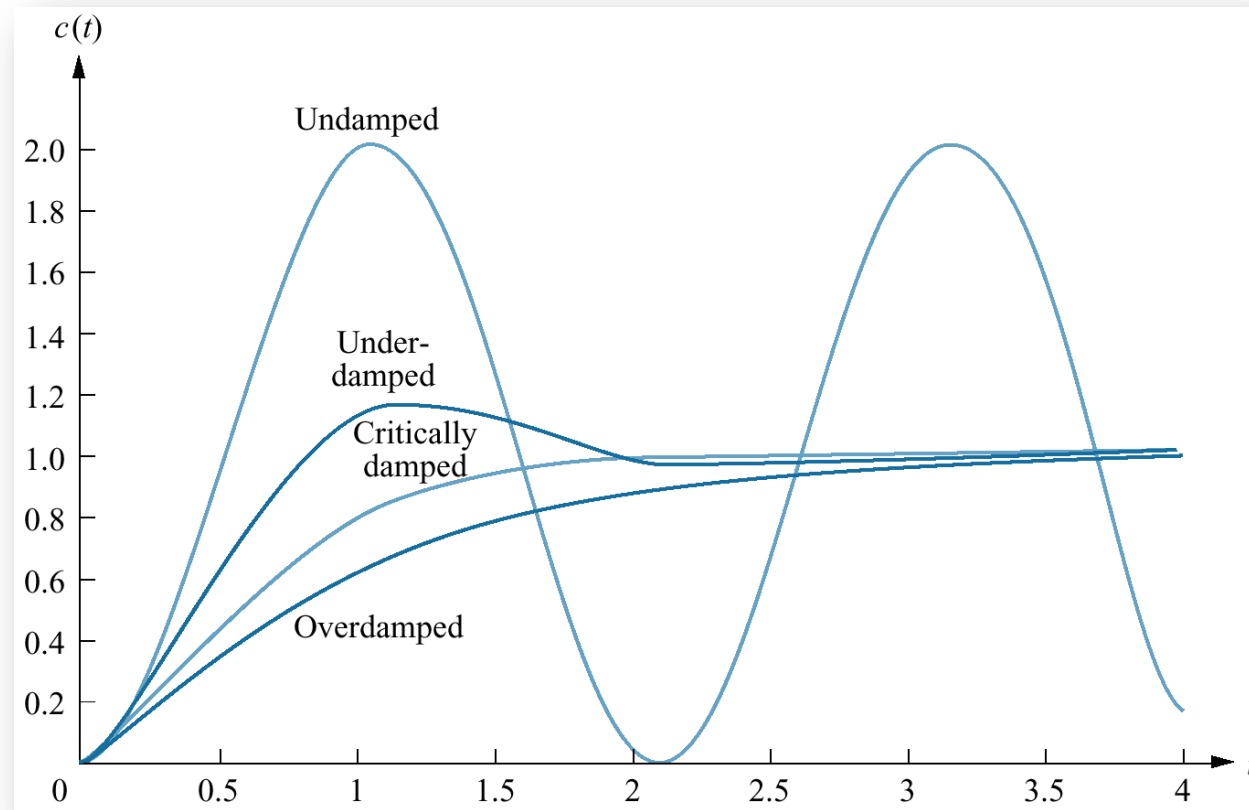


# Second-Order System



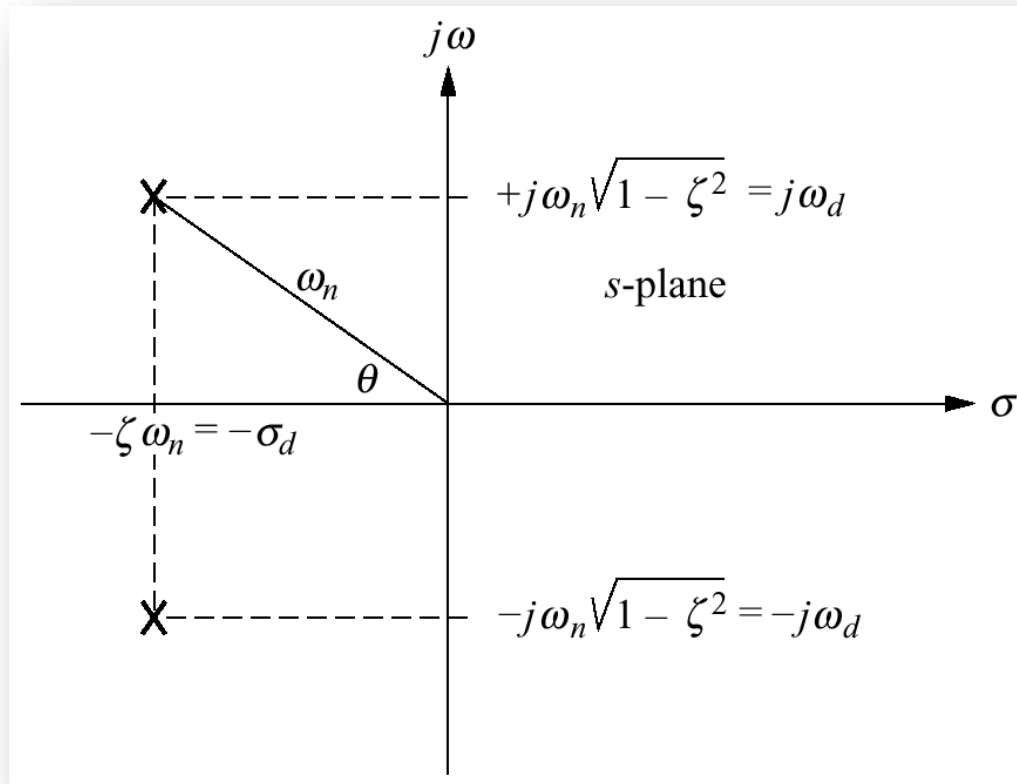
# Second-Order System

- ▶ Step responses for second-order system damping cases



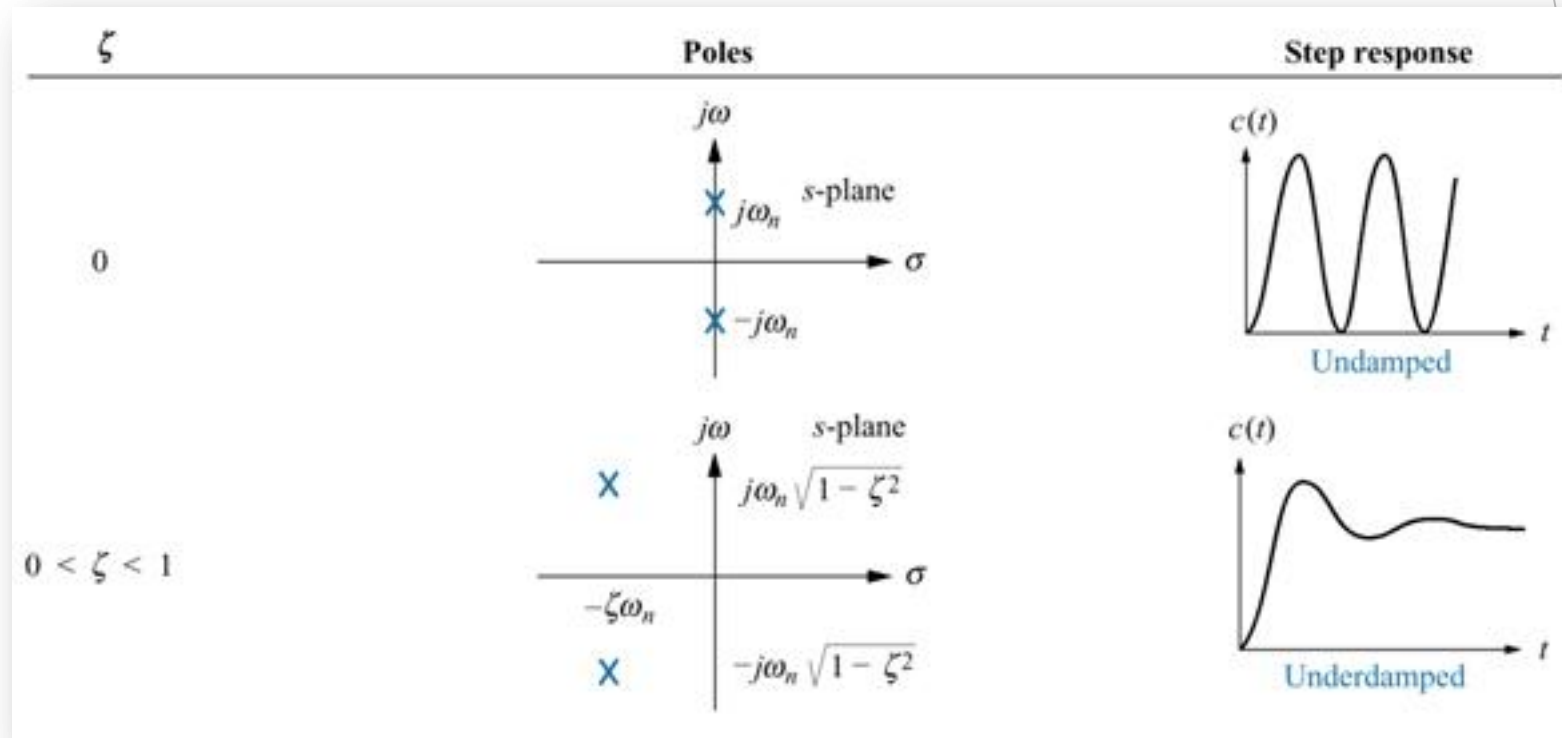
# Second-Order System

- Pole plot for the underdamped second-order system



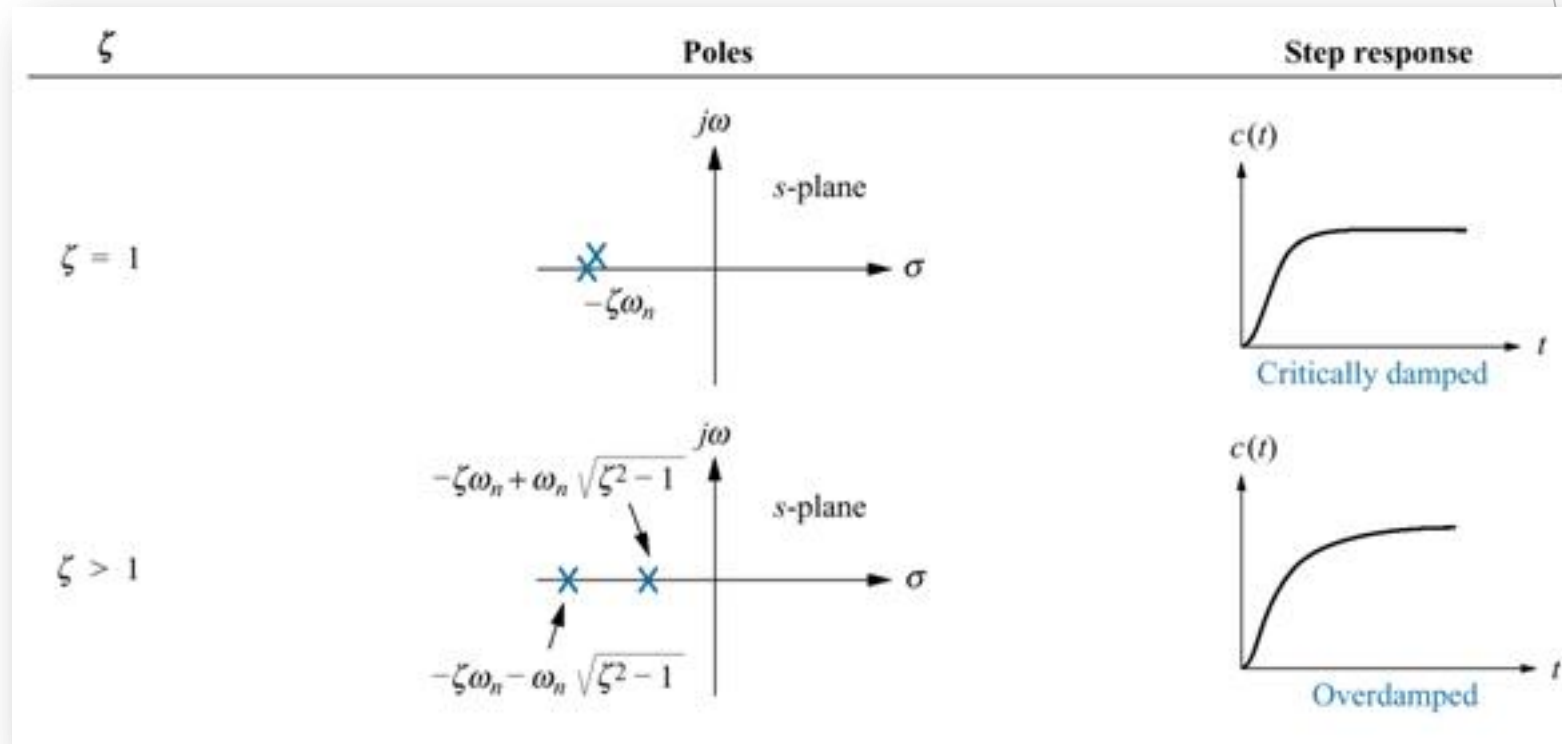
# Second-Order System

- ▶ Second-order response as a function of damping ratio



# Second-Order System

- ▶ Second-order response as a function of damping ratio



# Second-Order System

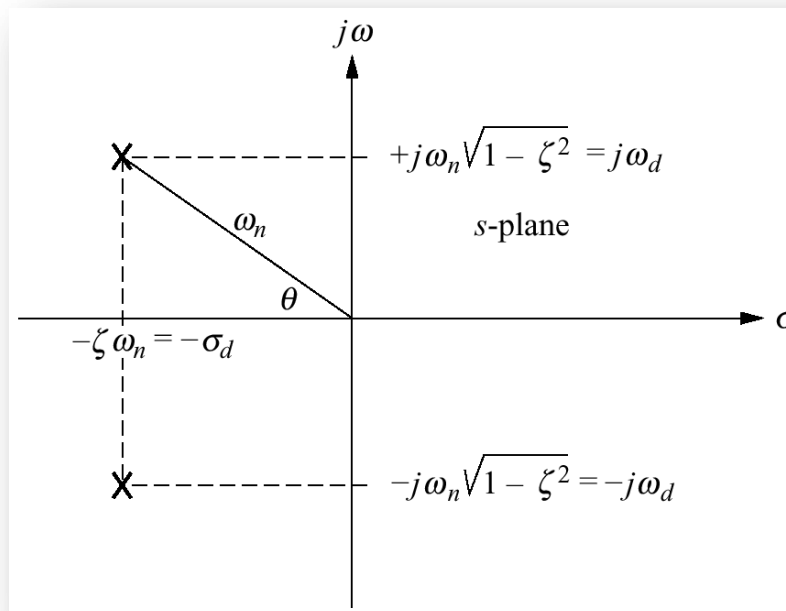
- ▶ When  $0 < \zeta < 1$ , the transfer function is given by the following.

$$G(s) = \frac{K\omega_n^2}{(s + \zeta\omega_n + j\omega_d)(s + \zeta\omega_n - j\omega_d)}$$

Where,

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

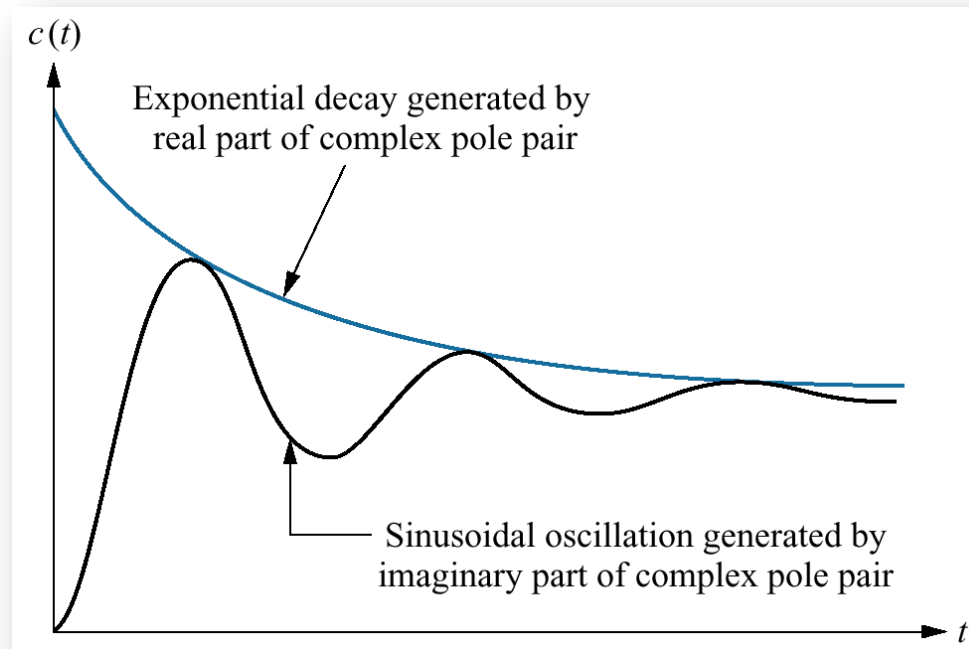
- ▶ Pole position:





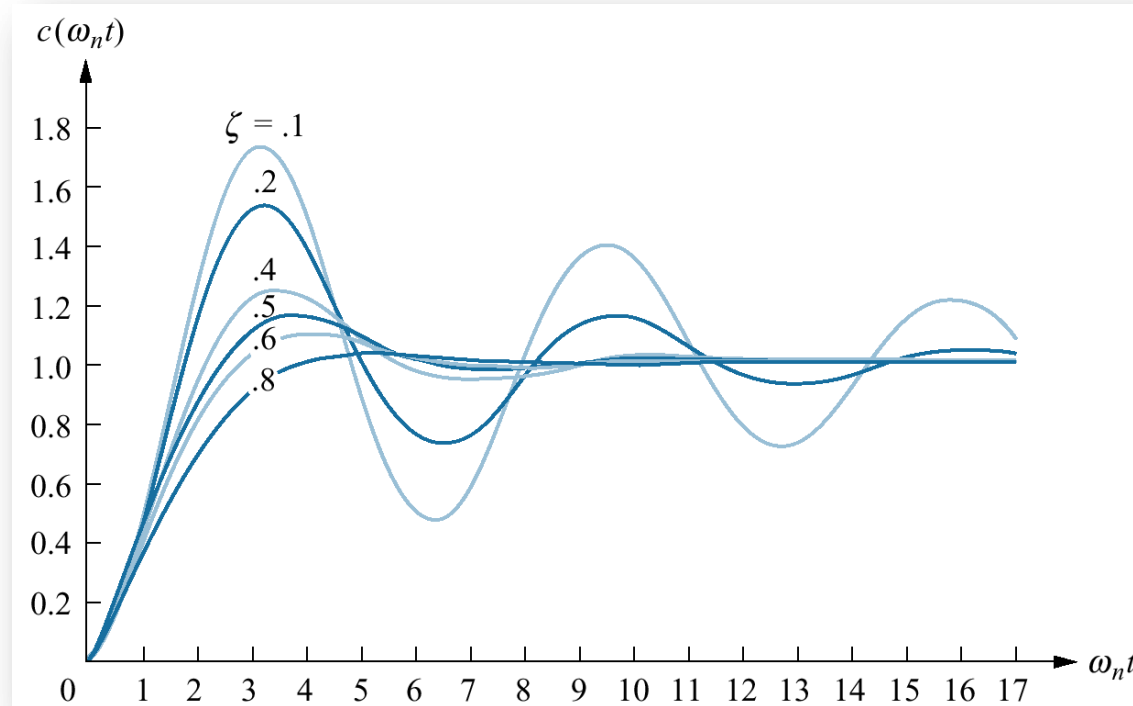
# Second-Order System

- ▶ Second-order response components generated by complex poles



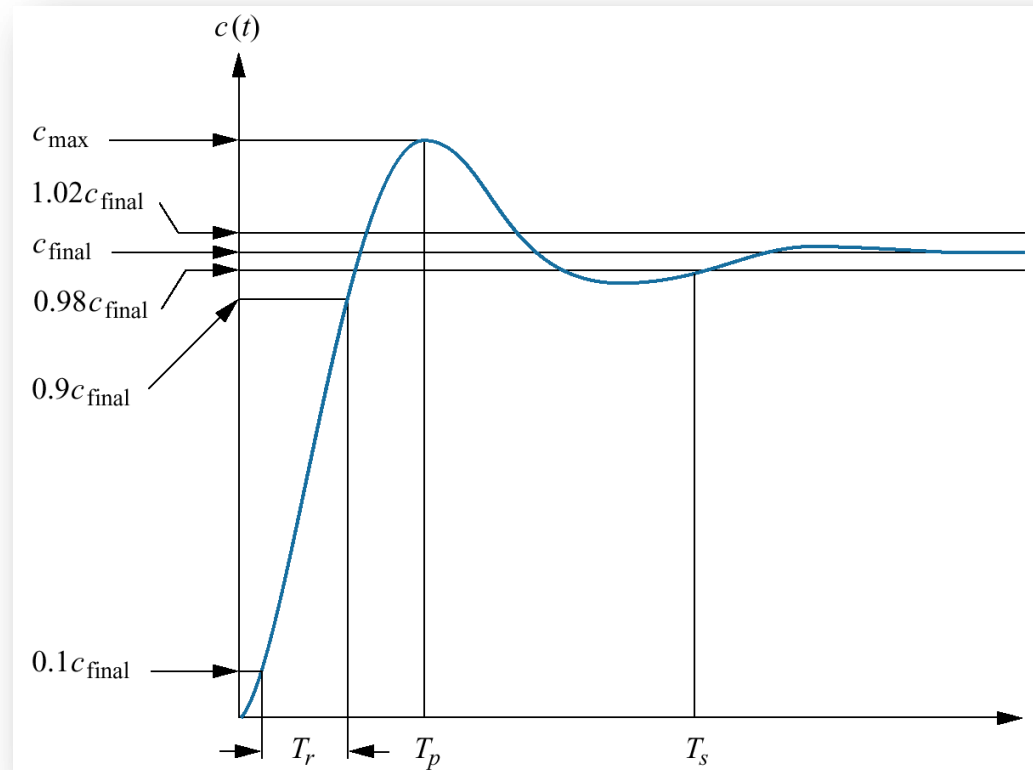
# Second-Order System

- ▶ Second-order underdamped responses for damping ratio value



# Transient Response Specifications

- ▶ Second-order underdamped response specifications



# Transient Response Specifications

- ▶ Rise time,  $T_r$

- ▶ The time for the waveform to go from 0.1 to 0.9 of its final value.

- ▶ Peak time,  $T_p$

- ▶ The time required to reach the first or maximum peak.

$$T_p = \frac{\pi}{\omega_n \sqrt{1 - \zeta^2}}$$

- ▶ Settling time,  $T_s$

- ▶ The time required for the transient's damped oscillation to reach and stay within  $\pm 2\%$  of the steady-state value.

$$T_s = \frac{4}{\zeta \omega_n}$$

# Transient Response Specifications

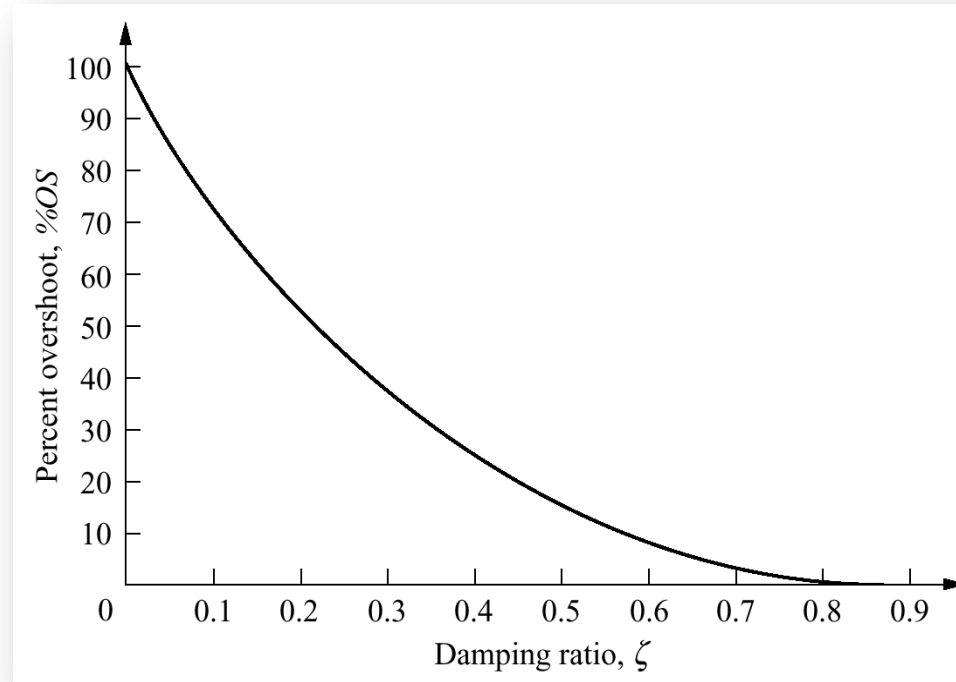
- ▶ Percent overshoot, %OS
  - ▶ The amount that the waveform overshoots the steady-state, or final value at peak time, expressed as a percentage of the steady-state value.

$$\%OS = e^{-(\zeta\pi/\sqrt{1-\zeta^2})} \times 100\%$$

$$\zeta = \frac{-\ln(\%OS/100)}{\sqrt{\pi^2 + \ln^2(\%OS/100)}}$$

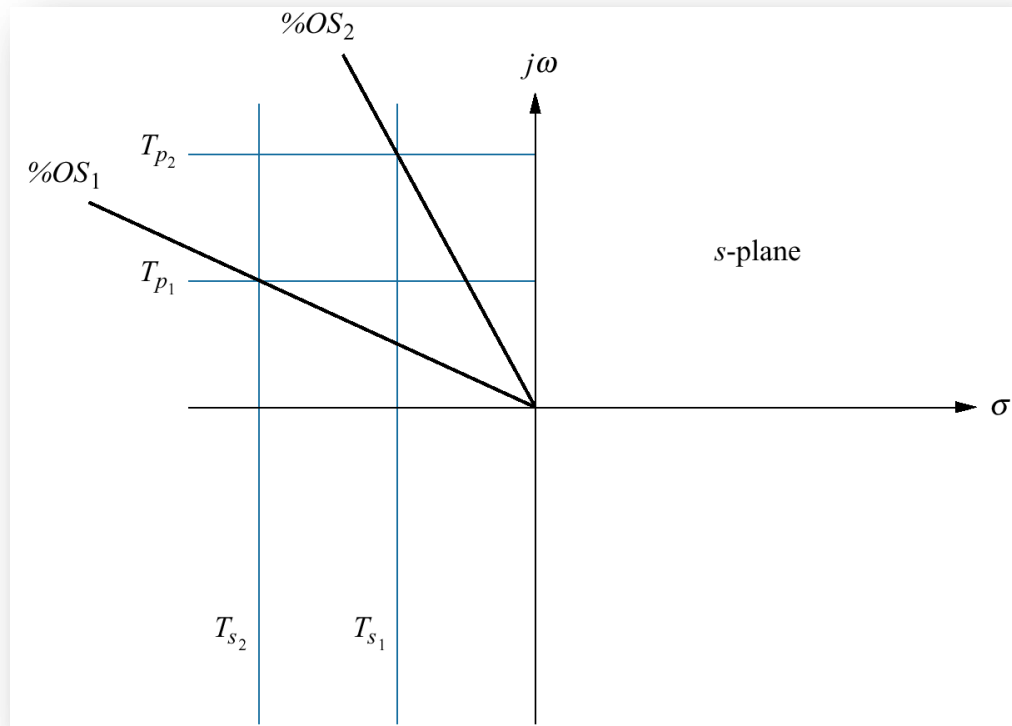
# System Performance

- ▶ Percent overshoot versus damping ratio



# System Performance

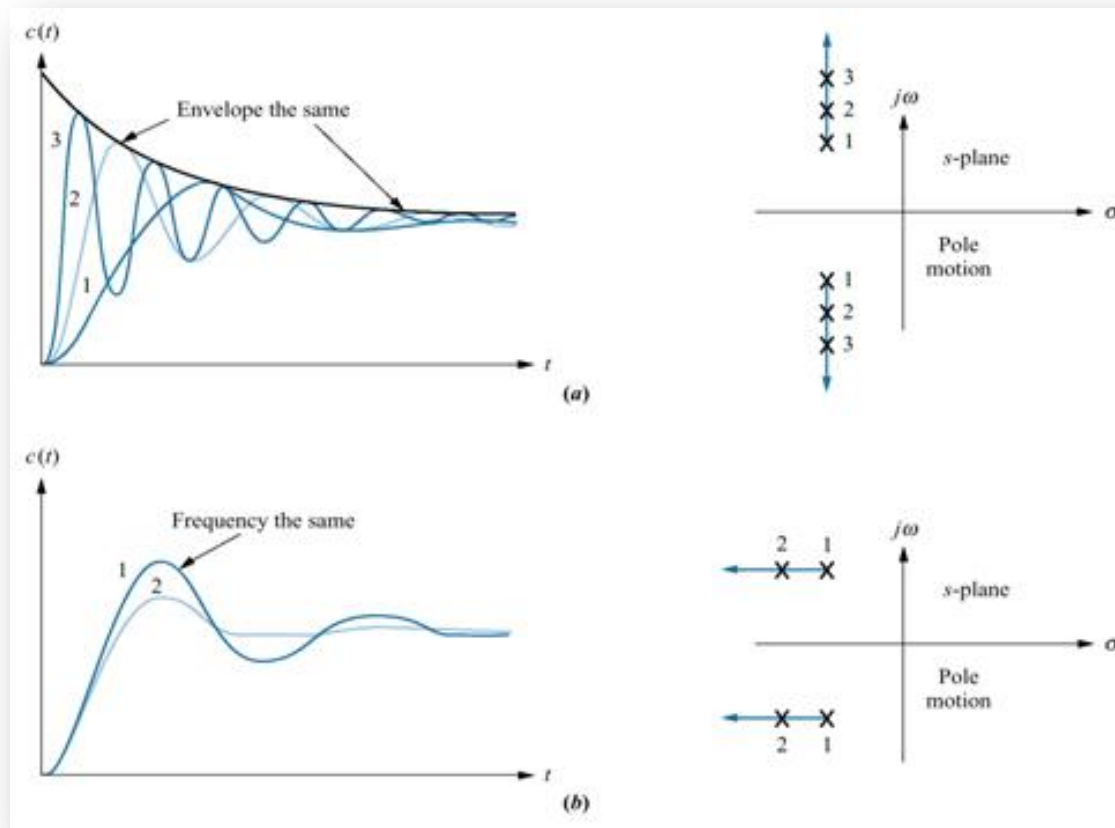
- ▶ Lines of constant peak time  $T_p$ , settling time  $T_s$  and percent overshoot %OS



$$\begin{aligned} T_{s2} &< T_{s1} \\ T_{p2} &< T_{p1} \\ \%OS_1 &< \%OS_2 \end{aligned}$$

# System Performance

- ▶ Step responses of second-order underdamped systems as poles move

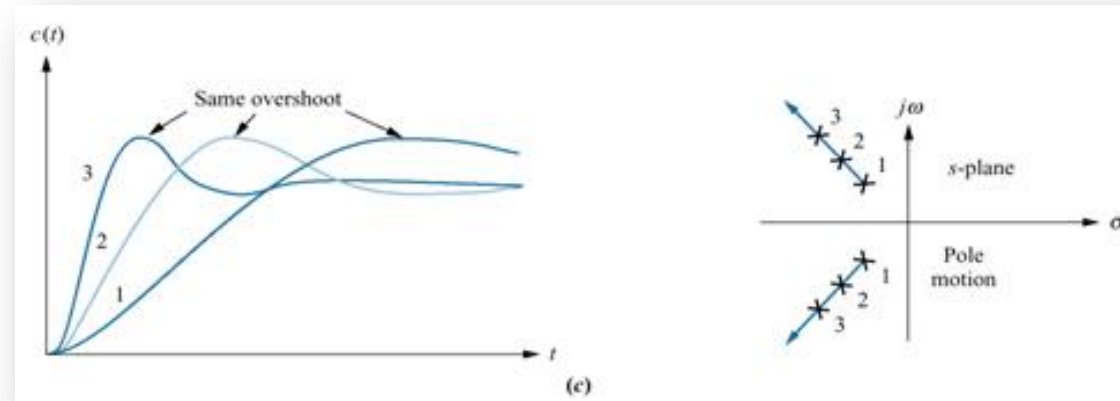


- a) With constant real part
- b) With constant imaginary part



# System Performance

- ▶ Step responses of second-order underdamped systems as poles move



- c) With constant damping ratio

# Further Reading...

## ▶ Chapter 4

- i. Nise N.S. (2004). Control System Engineering (4th Ed), John Wiley & Sons.

## ▶ Chapter 5

- i. Dorf R.C., Bishop R.H. (2001). Modern Control Systems (9th Ed), Prentice Hall.